



**Aalto University**  
School of Science  
and Technology

# **Advanced Wall Treatment Method for Turbulent Flow CFD Simulations**

**STAR European Conference**

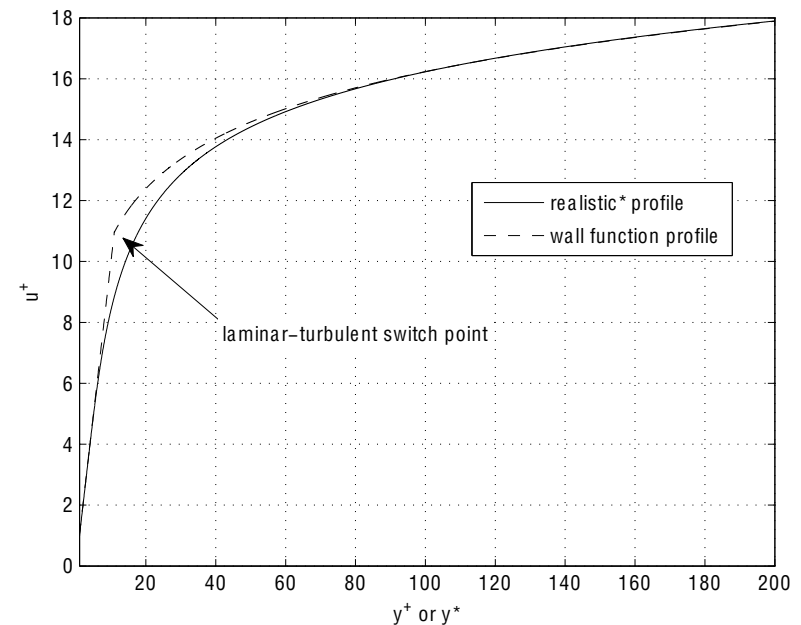
**Amsterdam, 22-23 March, 2011**

**Mika Nuutinen**

**Aalto University School of Science and Technology**

# NEAR WALL HEAT AND MOMENTUM TRANSFER

- ✓ **Difficult to measure or simulate**
- ✓ **Enormous gradients (T, U)**
- ✓ **Strong variation of ( $\rho$ ,  $\mu$ ,  $\kappa$ ,  $c_p$ ) with T**
- ✓ **Anisotropic near wall turbulence**



# HEAT (MOMENTUM) TRANSFER IN CFD

✓ Direct Numerical Simulation (DNS)

→ Accurate but too heavy for most engineering simulations

✓ Large Eddy Simulation (LES)

→ Still too heavy for accurate near wall modeling

✓ Reynold Averad Navier-Stokes (RANS)

→ Economical but inaccurate with standard methods

# NEAR WALL RANS MODELING OPTIONS

- ✓ **Low-Re/Two-layer models**  
→ require extensively fine mesh
- ✓ **High-Re models with wall functions**  
→ huge property variations, e.g.  $\rho(T)$ , impair accuracy
- ✓ **Advanced wall treatment/turbulence model** →  
→ Less restrictions, high accuracy!!!

# ADVANCED WALL TREATMENT/ TURBULENCE MODEL

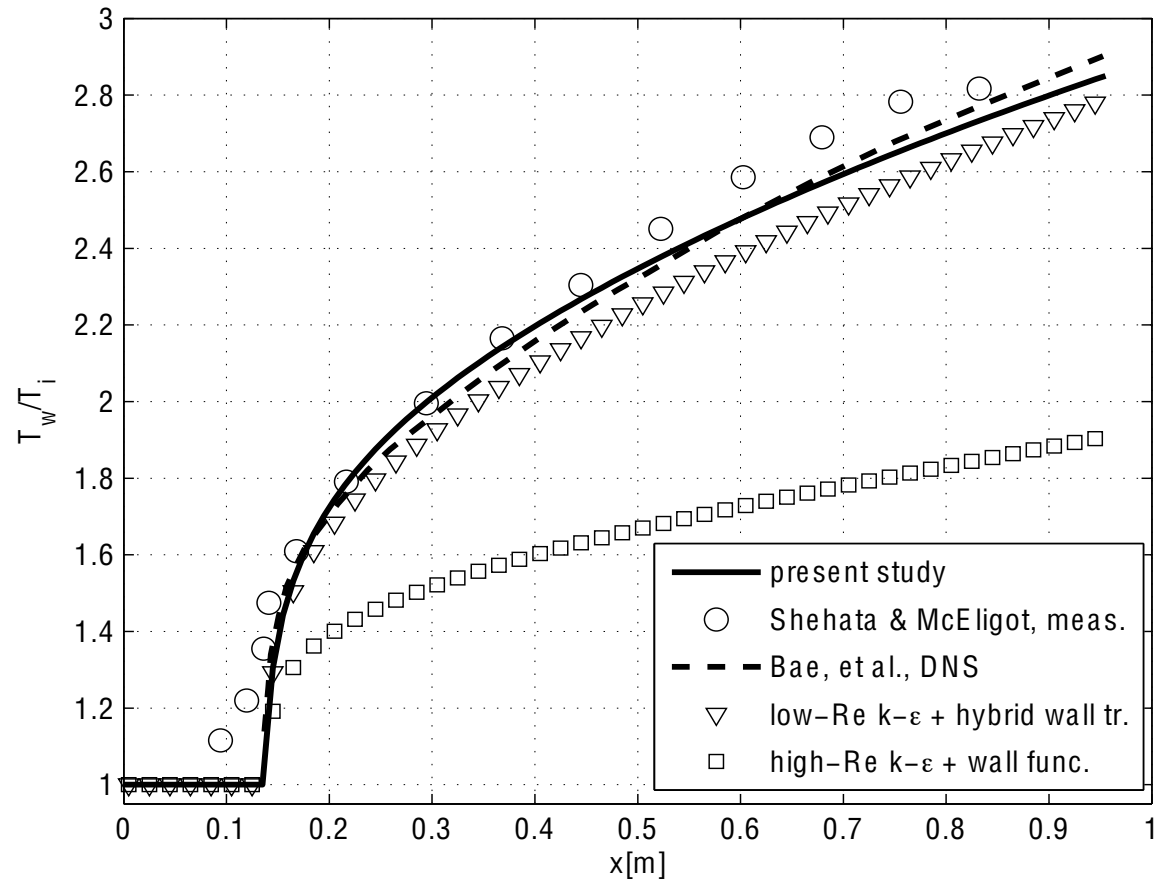
- ✓ **Based on a Low-Re  $k$ - $\epsilon$  turbulence model**
- ✓ **Relevant profiles discretized in wall adjacent cells, integrated numerically**
- ✓ **Discretization ADAPTIVE to local conditions!!!**
- ✓ **Accounts for  $\text{grad}(T)$  induced property variations ( $\rho$ ,  $\mu$ ,  $k$ ,  $c_p$ )**

# ADVANCED WALL TREATMENT/ TURBULENCE MODEL

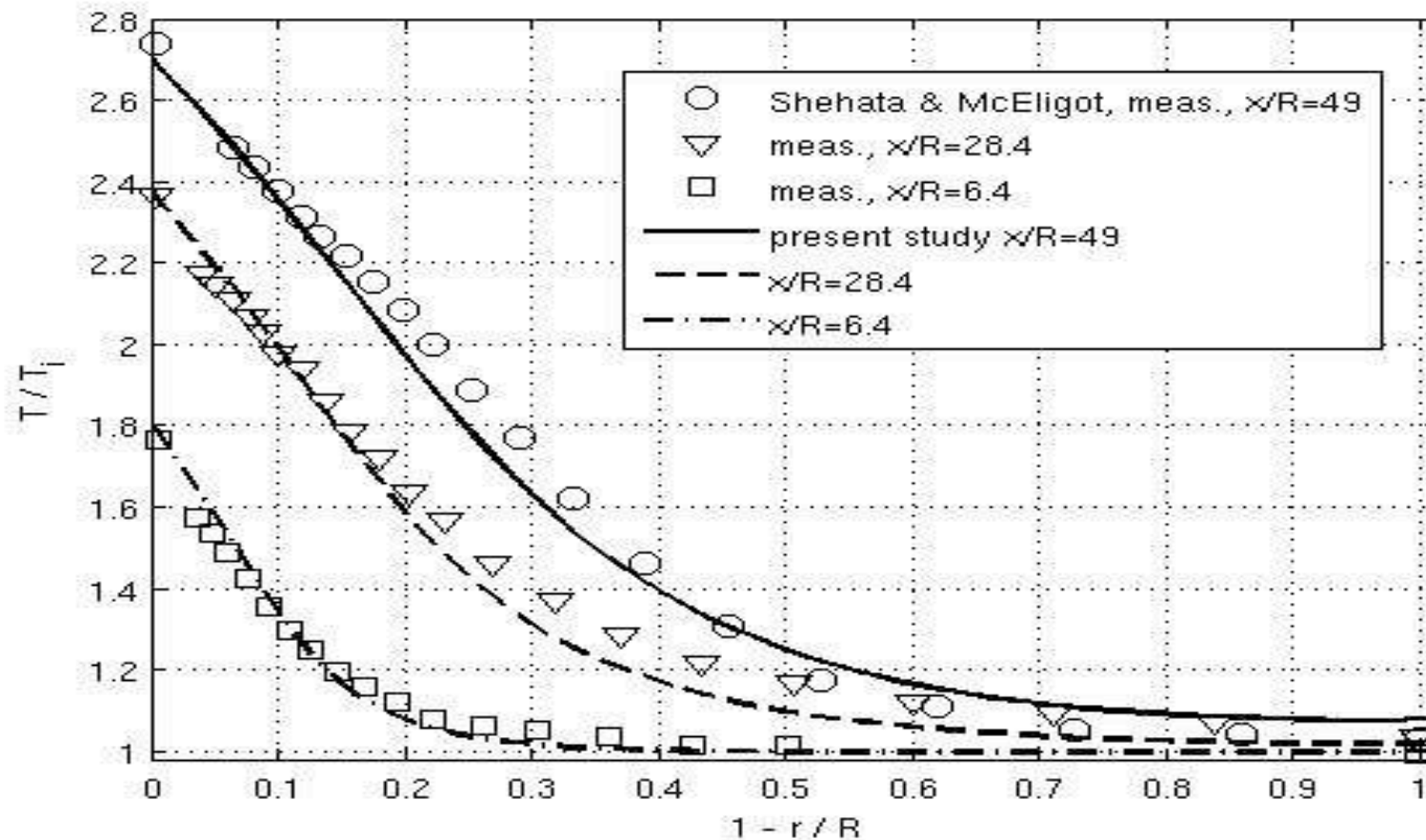
- ✓ **Enhanced accuracy in near wall turbulence source computation with any resolution (utilizing the discrete profiles)**
- ✓ **Free from the usual High/Low Re model grid spacing requirements**

# VALIDATION AGAINST MEAS. AND DNS-DATA, WALL TEMPERATURE

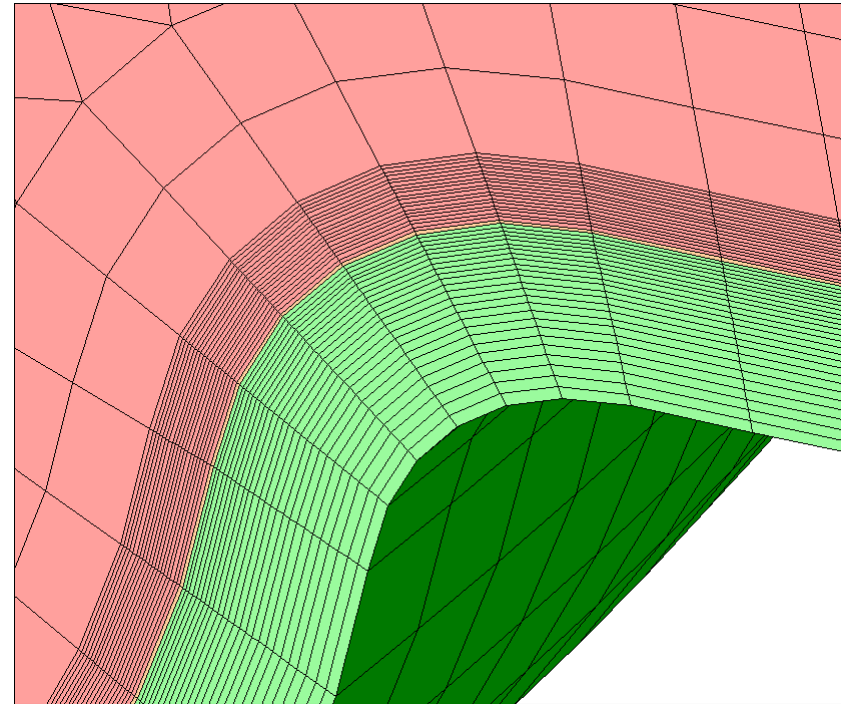
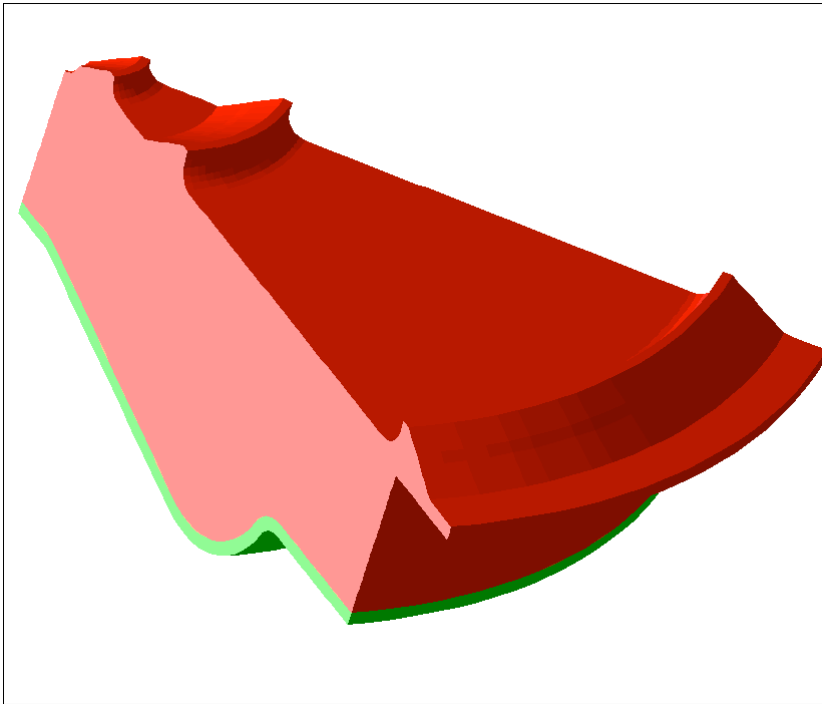
- Pipe Flow
- $Re=4300$
- $D=27.4\text{mm}$
- $L=0.96\text{m}$
- $q=3980\text{W/m}^2$
- $y^+ < 1$



# VALIDATION AGAINST MEAS., $T(r)$

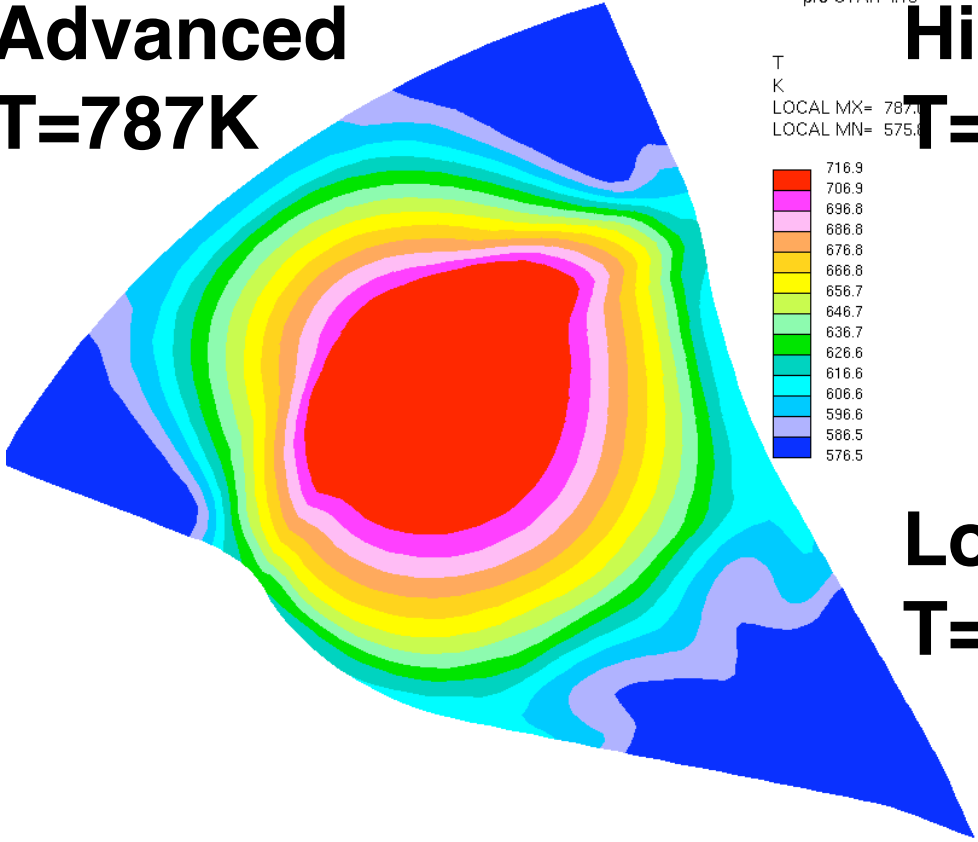


# CI ENGINE HEAT TRANSFER SIMULATIONS, 3D GRID

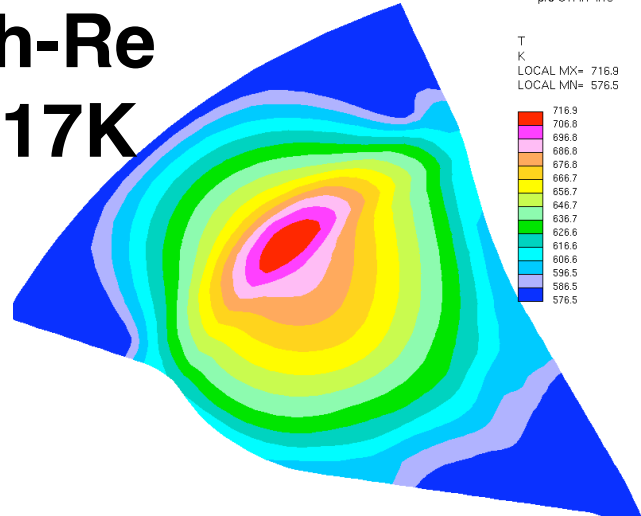


# CI ENGINE HEAT TRANSFER SIMULATIONS, SURFACE TEMP.

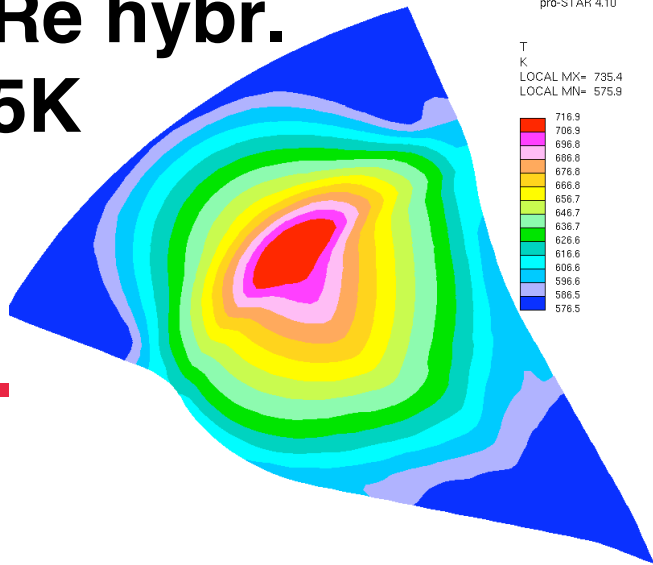
**Advanced**  
**T=787K**



**High-Re**  
**T=717K**

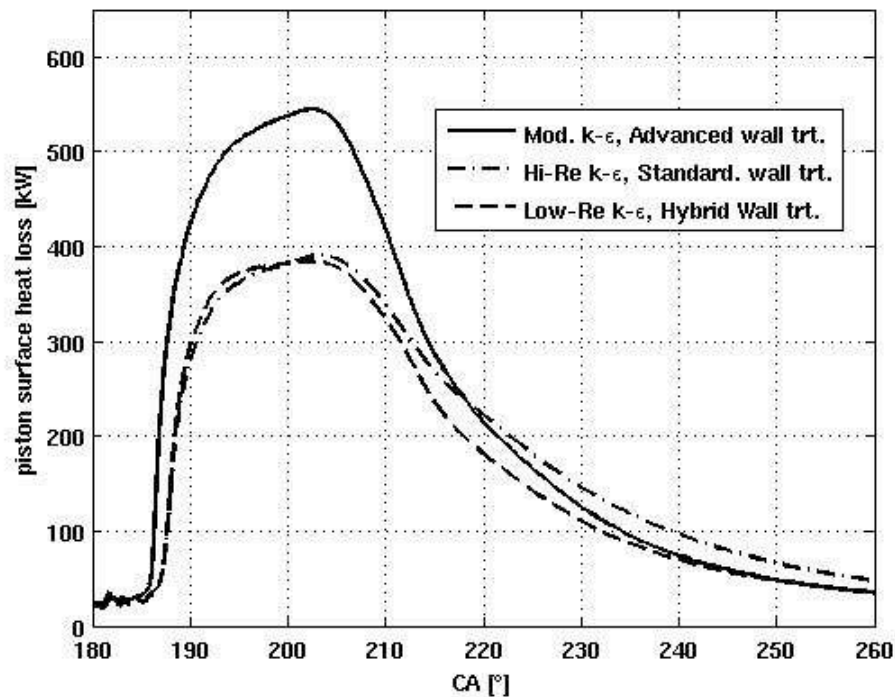


**Low-Re hybr.**  
**T=735K**

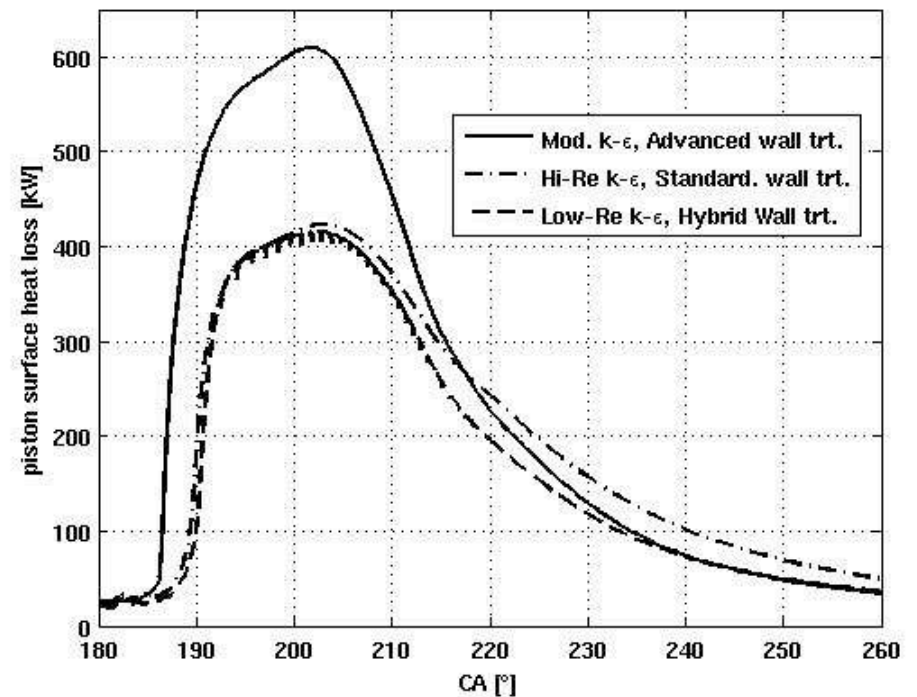


# TOTAL PISTON HEAT LOSS

## Conjugate Heat Transfer



## Const. Temp. B.C.



# CONCLUSIONS & FUTURE GOALS

- ✓ **Advanced model consistent with measurements and DNS**
- ✓ **Advanced model adaptive to grid resolution**
- ✓ **Advanced model successfully implemented on a commercial CFD code (Star-CD)**
- ✓ **Goals**
  - **Further model development**
  - **Piston temperature measurements**

# Near wall treatment equations

$$-\mu \frac{\partial^2 k}{\partial y^2} = \mu_t \left( \frac{\partial u}{\partial y} \right)^2 - \rho \varepsilon$$

$$\varepsilon = \tilde{\varepsilon} + \varepsilon_{wall} = \frac{C_{\mu}^{3/4} k^{3/2}}{\kappa y} + 2 \frac{\mu}{\rho} \frac{k}{y^2} =$$

$$\frac{C_{\mu}^{3/4} k^{3/2}}{\kappa y} \left( 1 + \frac{2\kappa C_{\mu}^{-3/4}}{\text{Re}_y} \right) =$$

$$\tilde{\varepsilon} \left( 1 + \frac{\alpha_{\varepsilon}}{\text{Re}_y} \right)$$

$$(\mu + \mu_t) \frac{\partial u}{\partial y} = \tau_w$$

$$c_p \left( \frac{\mu}{\text{Pr}} + \frac{\mu_t}{\text{Pr}_t} \right) \frac{\partial T}{\partial y} = q_w$$

$$\mu \frac{\partial^2 k}{\partial y^2} = 2\mu \frac{k}{y^2} \Big|_{y=0}$$

$$\frac{\rho C_{\mu} k^2 f_{\mu 1}}{\tilde{\varepsilon}} \left( \frac{\partial u}{\partial y} \right)^2 = \rho \tilde{\varepsilon}$$

$$f_{\mu 1} = \left( 1 - \exp(-\alpha_{\mu} \text{Re}_y) \right)$$

$$y^+ = \frac{\rho_w u_{\tau} y}{\mu_w}$$

$$u^+ = \frac{u}{u_{\tau}}$$

$$T^+ = \frac{\rho_w c_{p,w} u_{\tau} (T - T_w)}{q_w}$$

$$u_{\tau} = \sqrt{\tau_w / \rho_w}$$

$$\tau_w = \frac{y_c^+}{u_c^+} \mu_w \left\{ \frac{u_c - u_w}{y_c} \right\} =$$

$$\alpha^+ \mu_w \left\{ \frac{u_c - u_w}{y_c} \right\}$$

$$q_w = \frac{y_c^+}{T_c^+} c_{p,w} \mu_w \left\{ \frac{(T_c - T_w)}{y_c} \right\} =$$

$$\beta^+ c_{p,w} \mu_w \left\{ \frac{(T_c - T_w)}{y_c} \right\}$$

$$\frac{du^+}{dy^+} = \frac{2}{\lambda_1 + \sqrt{(\lambda_1^2 + 4\lambda_4 (\kappa y^+)^2 f_{\mu 1}^{3/2})}}$$

$$\frac{dT^+}{dy^+} = \frac{\text{Pr}_w \text{Pr}_t}{\lambda_2 \text{Pr}_t + \lambda_3 \text{Pr}_w \mu^+}$$

$$\mu^+ = \frac{\mu_t}{\mu_w} = \lambda_4 (\kappa y^+)^2 f_{\mu 1}^{3/2} \frac{du^+}{dy^+}$$

# Equations cont.

$$y_c^+ = \lambda_{1,c} \left( \frac{(y_c^*)^2 f_{\mu 1,c}^{1/2}}{\lambda_{4,c}} + \frac{y_c^*}{\lambda_{4,c} f_{\mu 1,c}^{1/2} \kappa} \right)^{1/2}$$

$$(\rho \varepsilon)_{avg} = \frac{\langle \chi_{dissip} \rangle_{avg}}{\chi_{dissip,c}} (\rho \varepsilon)_c$$

$$P^{wf} = \mu_w \mu^+ \left( \frac{\partial u^+}{\partial y^+} \right)^2 \left\{ \frac{y_c^+}{u_c^+} \frac{u_c - u_w}{y_c} \right\}^2$$

$$\chi_{dissip} = \left[ \mu_w \mu^+ + \mu \left( \frac{2\kappa^2}{C_\mu^{1/2}} \right) f_{\mu 1} \right] \left( \frac{\partial u^+}{\partial y^+} \right)^2$$

$$(\rho \varepsilon)^{wf} = \left[ \mu_w \mu^+ + \mu \left( \frac{2\kappa^2}{C_\mu^{1/2}} \right) f_{\mu 1} \right] \times$$

$$P_{avg,tot} = \left\{ P_1 + P_2 - \mu_{t,c} \left( \frac{\partial u_r^t}{\partial n} \right)^2 \right\} + P_{avg}$$

$$\left( \frac{\partial u^+}{\partial y^+} \right)^2 \left\{ \frac{y_c^+}{u_c^+} \frac{u_c - u_w}{y_c} \right\}^2$$

$$\frac{\partial u_r^t}{\partial n} = \langle \vec{n}, \vec{\nabla} u_r^t \rangle = t_i n_j \frac{\partial u_i}{\partial x_j}$$

$$P_{avg} = \langle \chi_{prod} \rangle_{avg} \left\{ \frac{u_c - u_w}{y_c} \right\}^2$$

$$\chi_{prod} = \mu_w \mu^+ \left( \frac{y_c^+}{u_c^+} \frac{\partial u^+}{\partial y^+} \right)^2$$

**THANK YOU!**

**QUESTIONS AND COMMENTS ARE  
WELCOME!**